

Extend the idea to 2D

$$1D: m\ddot{q} + c\dot{q} + kq = F$$

2D:

$$\begin{bmatrix} m_{11} & m_{12} \\ m_{21} & m_{22} \end{bmatrix} \begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} + \begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix} \begin{bmatrix} \dot{q}_1 \\ \dot{q}_2 \end{bmatrix} + \begin{bmatrix} k_{11} & k_{12} \\ k_{21} & k_{22} \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \end{bmatrix} = \begin{bmatrix} F_1 \\ F_2 \end{bmatrix}$$

$$F = -k_p q - k_d \dot{q}$$

$$\begin{bmatrix} F_1 \\ F_2 \end{bmatrix} = - \underbrace{\begin{bmatrix} k_{p11} & k_{p12} \\ k_{p21} & k_{p22} \end{bmatrix}}_{4 \text{ parameters}} \begin{bmatrix} q_1 \\ q_2 \end{bmatrix} - \underbrace{\begin{bmatrix} k_{d11} & k_{d12} \\ k_{d21} & k_{d22} \end{bmatrix}}_{4 \text{ parameters}} \begin{bmatrix} \dot{q}_1 \\ \dot{q}_2 \end{bmatrix}$$

8 parameters

To have a critical damped system
 ξ_1, ξ_2 there will be 2 equations.

Free parameters = $8 - 2 = \underline{6}$ (Too many)

Feedback Linearization / Control Partitioning

Feedback Linearization / Control partitioning

$$q \in \mathbb{R}^n$$

→ Dynamics

$$\underbrace{M(q)\ddot{q}}_{n \times n} + \underbrace{(C(q,\dot{q})\dot{q}}_{n \times 1} + \underbrace{G(q)}_{n \times 1} = \underbrace{\tau}_{n \times 1} \quad \text{①}$$

→ Feedback Control:

$$\tau = \underbrace{M}_{-}(-k_p q - k_d \dot{q}) + \underbrace{C(q,\dot{q})\dot{q}}_{-} + \underbrace{G(q)}_{-} \quad \text{②}$$

q, \dot{q} - measured by onboard sensors.

Substitute ② in ①

$$\cancel{M(q)\ddot{q} + C(q,\dot{q})\dot{q} + G(q)} = \cancel{M(-k_p q - k_d \dot{q})} + \cancel{(C(q,\dot{q})\dot{q} + G(q))} + \tau$$

$$M[\ddot{q} + k_p q - k_d \dot{q}] = 0 \quad \leftarrow$$

Since $M \neq 0$

$$\underbrace{\ddot{q} + k_p q - k_d \dot{q}}_{=} = 0$$

$$\ddot{q} + k_p q + k_d \dot{q} = 0$$

$$\begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \\ \ddot{q}_3 \\ \vdots \\ \ddot{q}_n \end{bmatrix} + \begin{bmatrix} k_{p_1} & 0 & 0 & \cdots & 0 \\ 0 & k_{p_2} & 0 & \cdots & 0 \\ 0 & 0 & k_{p_3} & \cdots & 0 \\ \vdots & & & & \vdots \\ 0 & 0 & \cdots & & k_{p_n} \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ \vdots \\ q_n \end{bmatrix} + \\ + \begin{bmatrix} k_{d_1} & 0 & 0 & \cdots & 0 \\ 0 & k_{d_2} & 0 & \cdots & 0 \\ 0 & 0 & k_{d_3} & \cdots & 0 \\ \vdots & & & & \vdots \\ 0 & 0 & \cdots & & k_{d_n} \end{bmatrix} \begin{bmatrix} \dot{q}_1 \\ \dot{q}_2 \\ \dot{q}_3 \\ \vdots \\ \dot{q}_n \end{bmatrix} = 0$$

These are n-decoupled equations

$$\left. \begin{array}{l} \ddot{q}_1 + k_{p_1} q + k_{d_1} \dot{q} = 0 \\ \ddot{q}_2 + k_{p_2} q + k_{d_2} \dot{q} = 0 \\ \vdots \\ \ddot{q}_n + k_{p_n} q + k_{d_n} \dot{q} = 0 \end{array} \right\} \underbrace{\ddot{q}_i + k_{p_i} q + k_{d_i} \dot{q} = 0}_{i=1, \dots, n}$$

$$m\ddot{q} + (c + k_d)\dot{q} + (k + k_p)q = 0 \quad \textcircled{1}$$

$$k_d = -c + 2\sqrt{(k+k_p)m}$$

~ 1 dof

$$\ddot{q}_i + k_{d,i}\dot{q}_i + k_{p,i}q_i = 0 \quad \textcircled{1^*}$$

$$m=1 \quad c=0 \quad k=0 \quad \text{in } \textcircled{1} \rightarrow \textcircled{1^*}$$

$$k_{d,i} = -c + 2\sqrt{(k+k_{p,i})m}$$

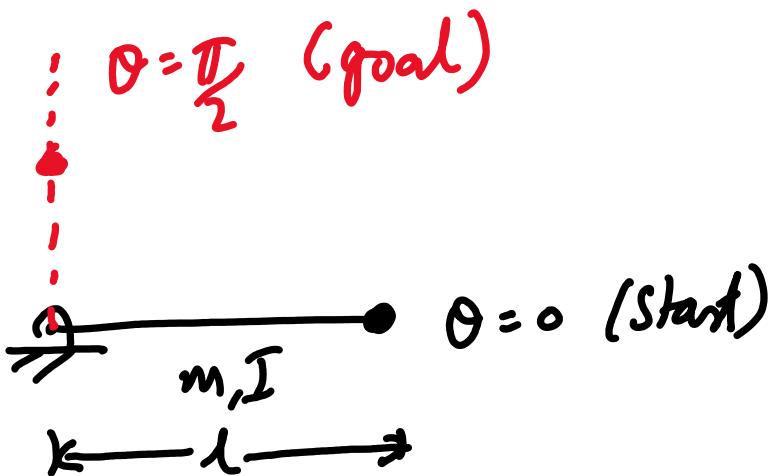
$$k_{d,i} = 0 + 2\sqrt{(0+k_{p,i})} i$$

$$k_{d,i} = 2\sqrt{k_{p,i}}$$

$i=1, 2, \dots, n$

{ See python file
 2-control_partitioning-pd }

EXAMPLE: 1-link pendulum



$$\begin{aligned} M(q) \ddot{q} + C(q, \dot{q}) \dot{q} + G(q) &= \tau \\ \hookrightarrow (I + ml^2) \ddot{q} + 0 + mgl \sin(q) &= \tau \end{aligned}$$

Controllers

Π_h

$$\textcircled{1} \quad \tau = -k_p (q - q_{des}) - k_d \dot{q}$$

3a - pendulum - pd

$$\textcircled{2} \quad \tau = M (-k_p (q - q_{des}) - k_d \dot{q}) + mgl \sin(q)$$

3b - pendulum - control - partitioning.

$$k_d = 2\sqrt{k_p}$$