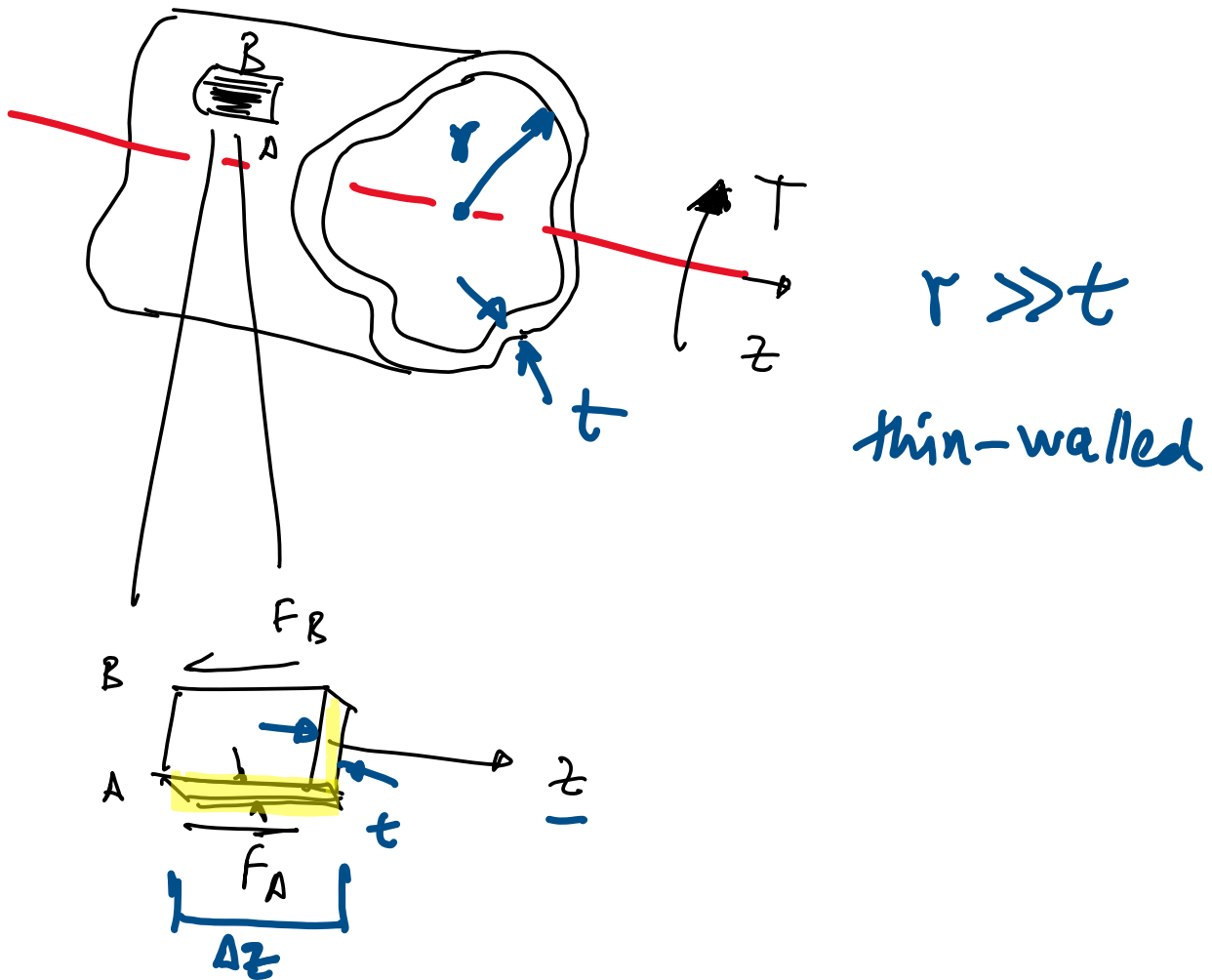


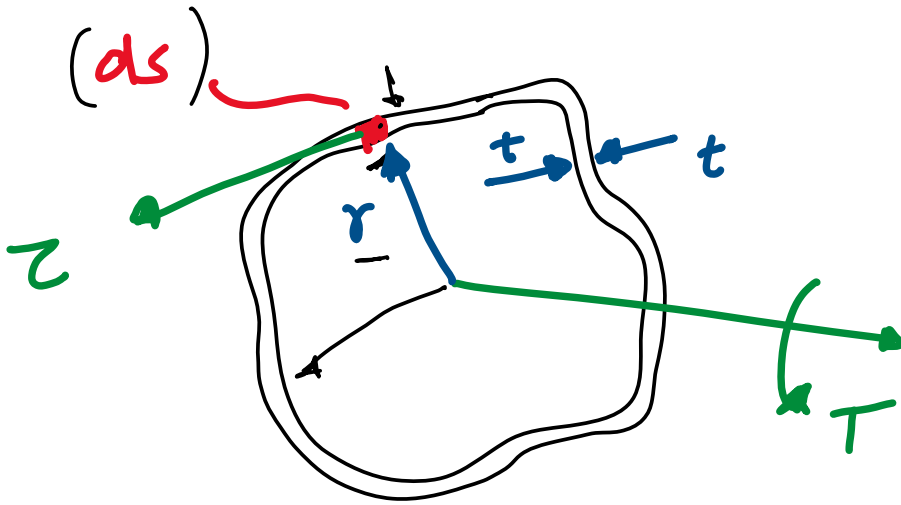
Shear stresses in a thin-walled tube



$$\begin{aligned} \sum F_z &= \underline{F_A} - \underline{F_B} \\ &= \underline{\tau_A} (\underline{t_A} \Delta z) - \tau_B (t_B \Delta z) = 0 \end{aligned}$$

$$\tau_A t_A = \tau_B t_B$$

$$\tau t = \text{constant}$$



$$dF = z dA = z(ds t)$$

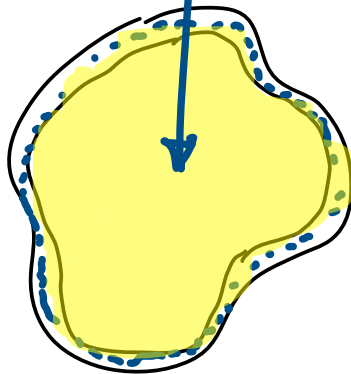
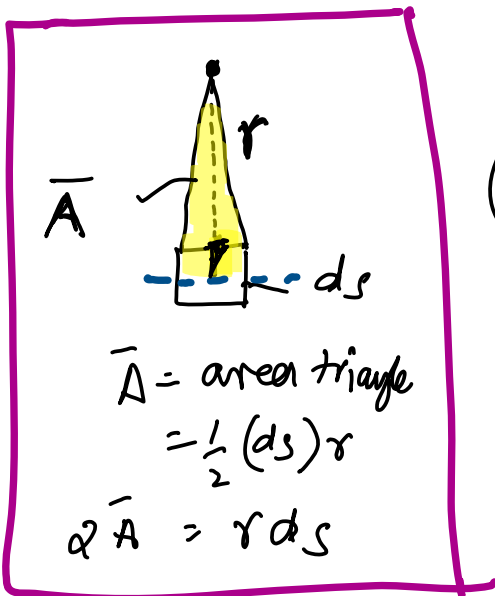
$$T = \int r dF = \int r z ds t$$

$$T = t z \int r ds$$

Why?

$$T = t z (\bar{z} A)$$

area enclosed by the median line (dashed line)

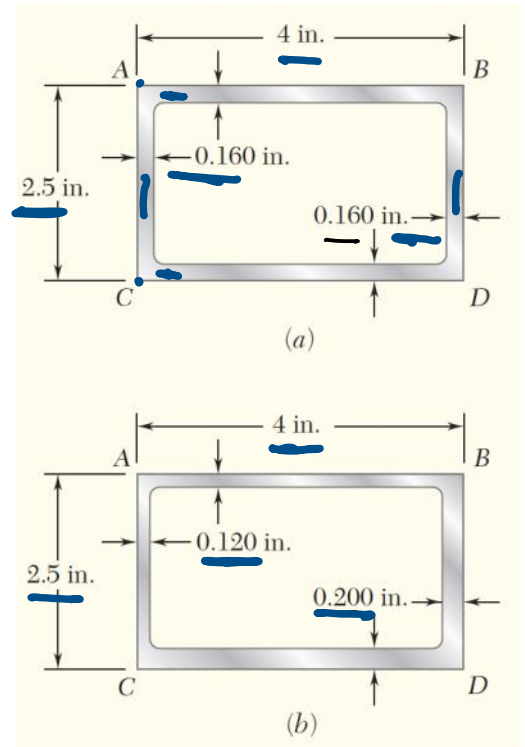
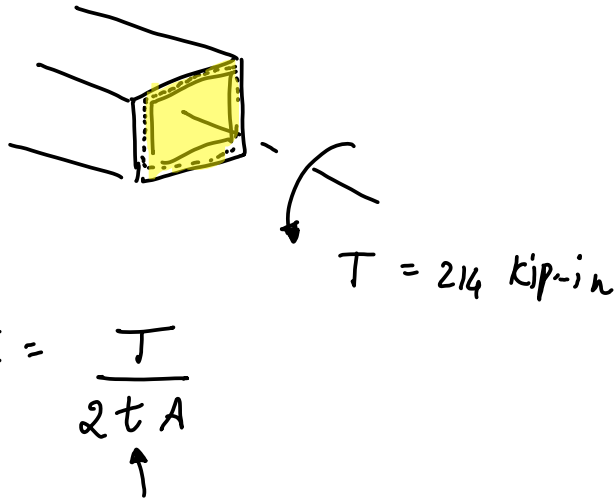


$$z = \frac{T}{2tA}$$

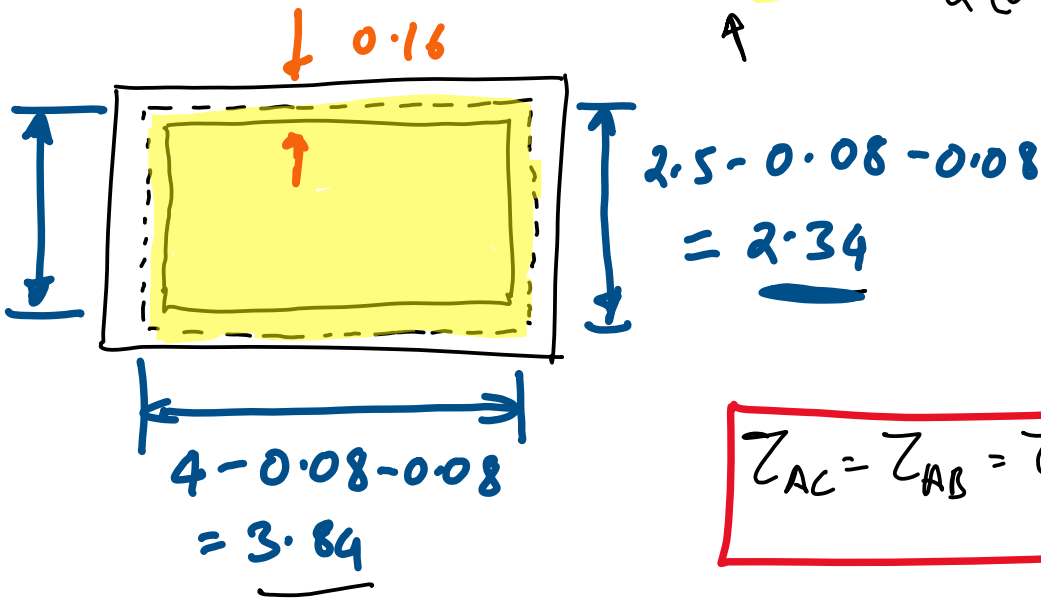
$$\theta = \frac{TL}{4GA^2t}$$

Q1

Compute the shearing stress in each of the four walls of two tubings when it is subjected to a torque of 24 kip-in.



(a) $\tau_{AC} = \tau_{AB} = \tau_{BD} = \tau_{DC} = \frac{T}{2 t A} = \frac{214}{2(0.16)(2.34)(3.84)}$



$$\tau_{AC} = \tau_{AB} = \tau_{BD} = \tau_{DC} = 8.35 \text{ ksi}$$

$$(b) \quad z_{AB} = z_{AC} = \frac{T}{2tA} = \frac{24}{2(0.12)(2.34)(3.84)} = 11.13 \text{ ksi}$$

$$z_{BD} = z_{DC} = \frac{T}{2tA} = \frac{24}{2(0.2)(2.34)(3.84)} = 6.68 \text{ ksi}$$

